Mathematical modeling of piezoceramic spherical shell actuator for vibration generating devices

Constantine Bazilo¹, Victor Antonyuk², Maksym Bondarenko³, Sergii Vysloukh⁴, **Oksana Voloshko5 , Roman Litvin6**

^{1, 3}Department of Instrumentation, Mechatronics and Computerized Technologies, Cherkasy State Technological University, boul. T. Shevchenka, 460, 18006, Cherkasy, Ukraine 2, 4, 5Department of Computer-Integrated Technologies of Device Production, National Technical University

of Ukraine, Igor Sikorsky Kyiv Polytechnic Institute, Beresteisksyi Prospect, 37, 03056, Kyiv, Ukraine ⁶Department of Motor Vehicle Transport, Lviv Polytechnic National University,

S. Bandery str., 32, 79013, Lviv, Ukraine

3Corresponding author

E-mail: ¹ *k.bazilo@chdtu.edu.ua*, 2*victor.antoniuk@gmail.com*, 3 *m.bondarenko@chdtu.edu.ua*, 4*vsp1@ukr.net*, 5 *voloshko_o@ukr.net*, 6*roman.h.litvin@lpnu.ua*

Received 21 August 2024; accepted 24 September 2024; published online 27 September 2024 DOI https://doi.org/10.21595/vp.2024.24470

Check for updates

69th International Conference on Vibroengineering in Lviv, Ukraine, September 26-29, 2024

Copyright © 2024 Constantine Bazilo, et al. This is an open access article distributed under the Creative Commons Attribution License, which permits unrestricted use, distribution, and reproduction in any medium, provided the original work is properly cited.

Abstract. The article examines the relevance of piezoceramic spherical shell actuators, which provide a high level of accuracy and stability in the generation of vibrational oscillations over a wide range of amplitude-frequency characteristics. The spherical shape of these actuators is chosen because it allows for uniform distribution of mechanical stresses across the actuator's surface, thereby enhancing its durability and efficiency compared to traditional flat or cylindrical designs. The study presents analytically derived mathematical models and formulas for determining the electrical impedance of piezoelectric elements and analyzes the dependence of the amplitude of electrical impedance on frequency and the charge ratio on the polarized surfaces of the spherical piezoceramic element. The proposed model enables the determination of electrical impedance as a function of cyclic frequency and various parameters (geometric, physical and mechanical, electrical) of the actuators. The study establishes that spherical shell actuators exhibit a higher quality factor (2-2.5 times greater than disk actuators and 1.4-1.8 times greater than cylindrical actuators) and a significant reduction in impedance (2.5-7 times lower) at electromechanical resonance frequencies around 2.4 kHz. Furthermore, spherical actuators demonstrate 20-30 % better resistance to deformation and 15-25 % lower susceptibility to local resonances compared to other actuator shapes, which enhances operational stability and vibration generation accuracy.

Keywords: mathematical model, piezoceramic actuator, spherical shell, vibration generating device.

1. Introduction

This article addresses the relevance of piezoceramic spherical shell actuators, which provide a high level of accuracy and stability in the process of generating vibrational oscillations across a wide range of amplitude-frequency characteristics. The spherical shape of these actuators is chosen because it allows for the uniform distribution of mechanical stresses across the actuator's surface, thereby enhancing its durability and efficiency compared to traditional flat or cylindrical designs. The study presents analytically derived mathematical models and formulas for determining the electrical impedance of piezoelectric elements and analyzes the dependence of impedance amplitude on frequency and the charge ratio on the polarized surfaces of the spherical piezoceramic element. The proposed model allows for the determination of electrical impedance as a function of cyclic frequency and various parameters (geometric, physical and mechanical, electrical) of the actuators.

The modern development of technologies related to vibration-generating devices primarily depends on the use of advanced materials and designs that ensure increased efficiency and reliability under various operating conditions. One such promising direction is the use of piezoceramic actuators, particularly those based on a spherical shell construction [1]. Mathematical modeling of such actuators is a key aspect of developing effective vibration-generating systems that have a wide range of applications in promising fields of healthcare and industry, such as medicine, pharmaceuticals, microelectronics, aerospace, and more.

The relevance of the research conducted in this article lies in the fact that piezoceramic spherical shell actuators provide a high level of accuracy and stability in the process of generating vibrational oscillations over a wide range of amplitude-frequency characteristics, which is critically important for most modern devices that require precise control of vibration parameters [2]. For example, the spherical shape of the shell allows for the even distribution of mechanical stresses across the entire surface of the actuator, which increases its durability and efficiency compared to traditional flat or cylindrical designs [3].

In the work [4], author Moon Kyu Kwak conducts dynamic modeling and develops control algorithms for digital controllers, including measurement methodologies and actuators. The article [5] analyzes the vibration response of a structure to external harmonic excitation, allowing for the study of control parameters to reduce tremor. Meanwhile, in [6], axisymmetric transverse bending vibrations of a bimorph piezoelement are investigated. However, as noted in works [7-10], there is currently no systematization of methods for the calculation and modeling of piezoceramic actuators, complicating the development of a generalized mathematical model. Specifically, for spherical shell actuators, where radially distributed vibrations are typically used, there is an urgent need to develop the theory of harmonic radial vibrations of piezoceramic cylindrical shells with limited height. Therefore, the development and experimental validation of mathematical models of piezoelectric spherical actuators for vibration-generating devices are currently relevant.

The aim of this article is to develop a mathematical model of a piezoceramic spherical shell actuator, which will allow for more accurate calculation of its electrical impedance, thus predicting its behavior under various load conditions and ensuring increased efficiency vibration-generating devices.

This scientific article contributes to the development of modern vibration technologies and may help improve the efficiency and reliability of vibration-generating systems in various fields of their application.

2. A mathematical model of a piezoceramic spherical shell

We calculate the electrical impedance $Z_{el}(\omega)$ of an oscillating open spherical piezoceramic shell (Fig. 1). Let us assume that the surfaces $\rho = R$ and $\rho = R + \alpha$ of the shell have electrodecoated using the combined technology thoroughly discussed in [11, 12]. The shell's material is polarized along the thickness, i.e., along the direction of the radial axis or in the spherical coordinate system (r, φ, ϑ) . In this case, obviously, the material constants of piezoceramics are determined by matrices [13].

Here we consider the frequency range in which the wavelength of elastic vibrations significantly exceeds the α thickness of the shell. If the shell vibrates in a vacuum, then the following statements are true:

$$
\sigma_{rr} = \sigma_{r\vartheta} = 0 \ \forall r \in [R, R + \alpha]. \tag{1}
$$

In the case of an axisymmetric, i.e., independent of the values of the azimuthal angle φ , stress-strain state of a spherical shell, in the general case, the following set of elastic stresses can exist in its volume and on its surface:

Fig. 1. An open spherical piezoceramic shell

Condition $\sigma_{rr} = 0 \,\forall r \in [R, R + \alpha]$ implies that:

$$
\varepsilon_{rr} = -\frac{c_{12}^E}{c_{11}^E} \left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} \right) + \frac{e_{11}}{c_{11}^E} E_r. \tag{6}
$$

Consequently:

$$
\sigma_{\phi\phi} = c_{22}\varepsilon_{\phi\phi} + c_{12}\varepsilon_{\vartheta\vartheta} - e_{12}^*E_r, \quad \sigma_{\vartheta\vartheta} = c_{12}\varepsilon_{\phi\phi} + c_{22}\varepsilon_{\vartheta\vartheta} - e_{12}^*E_r,\tag{7}
$$

where $c_{22} = c_{22}^E - (c_{12}^E)^2 / c_{11}^E$; $c_{12} = c_{12}^E (1 - c_{12}^E / c_{11}^E)$; $e_{12}^* = e_{12} - e_{11} c_{12}^E / c_{11}^E$ are material constants of piezoceramics for the mode of harmonic oscillations at constant (zero) normal voltage σ_{rr} .

The electrical axisymmetric state of a spherical shell is to be determined, in the general case, by the radial and polar components D_r and D_η of dielectric flux density vector. These components are defined as:

$$
D_r = e_{12}^* \left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} \right) + \chi_{11}^\sigma E_r,
$$

\n
$$
D_\vartheta = 2e_{35}\varepsilon_{r\vartheta} + \chi_{33}^\sigma E_\vartheta,
$$
\n(9)

where $\chi_{11}^{\sigma} = \chi_{11}^{\epsilon} + e_{11}^2/c_{11}^E$ is the dielectric piezoceramic constant in the mode of harmonic oscillations at constant (zero) normal stresses σ_{rr} .

Similarly to the case of a cylindrical shell [14], based on the condition $\sigma_{r\vartheta} = 0 \,\forall r [R, R + \alpha]$ we demonstrate that $\varepsilon_{r\vartheta} = 0$ and $E_{\vartheta} = 0$, which is equivalent to $D_{\vartheta} = 0$ $\forall r \in [R, R + \alpha]$.

Thus, the polarization effects (electrical state) in a strained spherical piezoceramic shell are determined by the radial component D_r of the dielectric flux density vector. Condition $div\vec{D} = 0$ implies that $\frac{1}{r^2} \left[\frac{\partial}{\partial r} (r^2 D_r) \right] = 0$ from whence we find $D_r = C_1 / r^2$, where C_1 is the constant to be defined.

Remembering that $E_r = -\partial \Phi(r)/\partial r$, where $\Phi(r)$ is the scalar potential of the electric field in the volume of strained piezoceramics, we obtain the equation:

MATHEMATICAL MODELING OF PIEZOCERAMIC SPHERICAL SHELL ACTUATOR FOR VIBRATION GENERATING DEVICES. CONSTANTINE BAZILO, VICTOR ANTONYUK, MAKSYM BONDARENKO, SERGII VYSLOUKH, OKSANA VOLOSHKO, ROMAN LITVIN

$$
e_{12}^*\left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta}\right) - \chi_{11}^\sigma \frac{\partial \Phi(r)}{\partial r} = \frac{C_1}{r^2}.\tag{10}
$$

Integrating the left and the right sides of Eq. (10), we obtain the expression:

$$
e_{12}^*\left(\varepsilon_{\phi\phi} + \varepsilon_{\partial\theta}\right)(R - r) - \chi_{11}^{\sigma}[\Phi(R) - \Phi(r)] = -C_1\left(\frac{1}{R} - \frac{1}{r}\right) + C_2,\tag{11}
$$

where C_2 is the integration constant.

Since $\Phi(R) = U_0$, and $\Phi(R + \alpha) = 0$ (Fig. 1), the constant $C_2 = 0$, while the C_1 constant is to be determined by the following expression:

$$
C_1 = R^2 \left(1 + \frac{\alpha}{R} \right) \left[e_{12}^* \left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} \right) + \chi_{11}^{\sigma} \frac{U_0}{\alpha} \right].
$$
 (12)

Thus, the radial component D_r of the dielectric flux density vector necessary to determine the electrical impedance $Z_{el}(\omega)$ of a spherical piezoceramic shell will be calculated by the formula:

$$
D_r = \frac{R^2}{r^2} \left(1 + \frac{\alpha}{R} \right) \left[e_{12}^* \left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} \right) + \chi_{11}^\sigma \frac{U_0}{\alpha} \right]. \tag{13}
$$

The electric charge Q on the surface $r = R$ defines as the integral over area. Since the area element is $dS = R^2 \sin \theta d\theta d\varphi$, the following implies:

$$
Q = 2\pi R^2 \left(1 + \frac{\alpha}{R}\right) \int_{\vartheta_0}^0 \left[e_{12}^* \left(\varepsilon_{\varphi\varphi} + \varepsilon_{\vartheta\vartheta}\right) + \chi_{11}^\sigma \frac{U_0}{\alpha}\right] \sin\vartheta d\vartheta. \tag{14}
$$

In the integrand Eq. (14) we consider a new variable $\xi = \cos \theta$. Then $\sin \theta d\theta = -d\xi$ and:

$$
Q = -2\pi R^2 \left(1 + \frac{\alpha}{R}\right) \int_{\cos\vartheta_0}^1 e_{12}^* \left(\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta}\right) d\xi - 2\pi R^2 \left(1 + \frac{\alpha}{R}\right) \left(1 - \cos\vartheta_0\right) \chi_{11}^\sigma \frac{U_0}{\alpha}.\tag{15}
$$

For an axisymmetric stress-strain state of the open spherical shell, the strain tensor components $\varepsilon_{\varphi\varphi}$ and $\varepsilon_{\vartheta\vartheta}$ are found by the following relations:

$$
\varepsilon_{\phi\phi} = \frac{1}{R} [u_r(\vartheta) + u_\vartheta(\vartheta) \text{ ctg}\vartheta], \quad \varepsilon_{\vartheta\vartheta} = \frac{1}{R} \bigg[u_r(\vartheta) + \frac{\partial u_\vartheta(\vartheta)}{\partial \vartheta} \bigg],
$$

where $u_r(\vartheta)$ and $u_{\vartheta}(\vartheta)$ are radial and polar components of the material particle displacement vector of an oscillating spherical shell.

Based on the found sensitivity of the piezoceramic shell in the inverse piezoelectric effect mode, i.e., assuming that $u_r(\theta) = U_0 f_r(\theta)$ and $u_{\theta}(\theta) = U_0 f_{\theta}(\theta)$, the equation reads as:

$$
\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} = \frac{U_0}{R} \bigg[2f_r(\vartheta) + \frac{\partial f_\vartheta(\vartheta)}{\partial \vartheta} + f_\vartheta(\vartheta) \operatorname{ctg}\vartheta \bigg]. \tag{16}
$$

Condition $\varepsilon_{r\vartheta} = 0$ $\forall r \in [R, R + \alpha]$ implies that if $\partial/\partial r = 0$, this condition is equivalent to the statement $\frac{\partial u_r(\theta)}{\partial \theta} - u_{\theta}(\theta) = 0$ $\forall r \in [R, R + \alpha]$, from whence it follows that $f_{\theta}(\theta) = \frac{\partial f_r(\theta)}{\partial \theta}$.

Using relation last equation to exclude the $f_{\vartheta}(\vartheta)$ function from the square bracket in formula

Eq. (16), we obtain the notation:

$$
\varepsilon_{\phi\phi} + \varepsilon_{\vartheta\vartheta} = \frac{U_0}{R} \bigg[2f_r(\vartheta) + \frac{\partial^2 f_r(\vartheta)}{\partial \vartheta^2} + \frac{\partial f_r(\vartheta)}{\partial \vartheta} \text{ctg}\vartheta \bigg]. \tag{17}
$$

Further, we introduce the notation $C_{ss}^{\sigma} = 2\pi R(1 + \alpha/R) (1 - \cos\theta_0) \chi_{11}^{\sigma}/\alpha$ and this quantity will be termed the dynamic electric capacitance of an open spherical piezoceramic shell. Counting this notation and definition Eq. (17) of the total strain, the expression Eq. (15) takes the following form:

$$
Q = -\mathcal{C}_{ss}^{\sigma} U_0[F_{ss}(\Gamma, \Pi) + 1],\tag{18}
$$

where $F_{\rm sc}(\Gamma,\Pi)$ is a dimensionless frequency-dependent function determined by a set of geometric (symbol Γ) and physical-mechanical (Π) parameters of a spherical open piezoceramic shell. The numeric values of the $F_{ss}(\Gamma,\Pi)$ function are found by the following expression:

$$
F_{ss}(\Gamma,\Pi)=\frac{\alpha e_{12}^*}{(1-\cos\vartheta_0)\chi_{11}^{\sigma}R}\int\limits_{\cos\vartheta_0}^1\left[2f_r(\vartheta)+\frac{\partial^2 f_r(\vartheta)}{\partial\vartheta^2}+\frac{\partial f_r(\vartheta)}{\partial\vartheta}\,\mathrm{ctg}\vartheta\right]d\xi.
$$

Since the amplitude value of the current is $I = -i\omega Q$, the electrical impedance of an open spherical shell is found (at this stage it is obvious) by the standard expression:

$$
Z_{el}(\omega) = \frac{U_0}{I} = \frac{1}{i\omega C_{ss}^{\sigma}[F_{ss}(\Gamma,\Pi)+1]}.
$$
\n(19)

Ultimately, Eq. (19) is valid for the high-frequency vibration range, where both radial and axial displacements of the material particles of the sphere occur simultaneously, meaning that the wavelength of the elastic wave becomes comparable to the thickness of the spherical actuator.

Let's examine the results of the mathematical modeling of the piezoceramic spherical shell actuator and analyze its electrical impedance in the high-frequency range.

At $e_{12}^* = 0$, function $F_{ss}(\Gamma, \Pi) = 0$, and expression Eq. (19) simplifies to the formula for calculating the reactance of a capacitor with electrical capacitance C_{ss}^{σ} .

It is important to note that the electrical impedance of the sphere is determined by the averaged values of the displacement vector components of the material particles, which leads to their transformation into ordinary differential equations that can be solved with a certain degree of accuracy. However, such a system of partial differential equations cannot be solved in its general form.

As for the medium and low-frequency ranges, the application of spherical shell piezoelectric actuators at these frequencies is impractical. Due to the peculiarities of the electroelastic state of the vibrating spherical transducer, at low frequencies, there is a negative change in the numerical values of the dynamic electrical capacitance and the analytical structure, thereby reducing the efficiency of such devices.

3. Discussion of simulation results

The above examples of the general definition for the electrical impedance of oscillating piezoceramic elements are sufficient to conclude that the fundamental condition $div\vec{D} = 0$ completely determines the electrical state of the objects under study. Moreover, fulfillment of $div\overline{D} = 0$ condition produces the definitions of the electric field strengths that appear in the generalized Hooke's law formulations, i.e., the $div\overline{D} = 0$ condition indirectly determines the

elastic stresses that arise in the volume and on the surface of oscillating piezoceramic elements. In other words, fulfillment of the fundamental condition $div\vec{D} = 0$ should precede any calculation of characteristics and parameters of oscillating piezoceramic elements [15]. This, perhaps, is one of the principles of mathematical modeling of the characteristics and parameters of oscillating piezoceramic elements [16].

Fig. 2 demonstrates the frequency-dependent variation of the dimensionless function $F_{\text{co}}(\omega)$, which was calculated for a set of geometric, physical, and mechanical parameters: $c_{11}^E = 110$ GPa; $c_{12}^E = 60$ GPa; $c_{33}^E = 100$ GPa; $e_{33} = 18$ C/m²; $e_{31} = -8$ C/m²; $\chi_{33}^E = 1400\chi_0$ (its parameters are characteristic of PZT-piezoceramics).

Fig. 2. Frequency-dependent alteration in the modulus of function $F_{co}(\omega)$

The inset in Fig. 2 shows the variation of the function's modulus in the range of dimensionless wave numbers close to the resonant values. The calculated value of this function enables the determination of the modulus of dynamic electrical capacitance in the spatial vibration mode of a piezoceramic spherical shell, and consequently, the assessment of the actuator's efficiency compared to actuators of other shapes. Spherical shell piezo actuators exhibit several positive effects compared to cylindrical and disk actuators [13-16], in particular:

1) Higher quality factor: due to their geometry, spherical shells can achieve a quality factor in the range of 500-800, significantly exceeding the values for cylindrical (300-500) and disk actuators (200-400).

2) Reduced electrical impedance: the spherical shape of the actuator helps lower electrical impedance to 10-20 Ohms, which is considerably lower than that of cylindrical (20-50 Ohms) and disk actuators (30-70 Ohms).

3) Wide range of resonant frequencies: spherical shell actuators can operate within a resonant frequency range from 1 kHz to 1 MHz, maintaining stable characteristics across different frequencies, while cylindrical and disk actuators typically operate between 10 kHz and 500 kHz.

4) Stability in operation: thanks to their spherical geometry, these actuators have 20-30 % better resistance to deformation, reducing the likelihood of unwanted oscillations.

5) Lower susceptibility to local resonances: spherical actuators are 15-25 % less prone to local resonances compared to disk and cylindrical actuators, which positively impacts the quality and precision of vibration generation.

4. Conclusions

In the course of the research, mathematical models were analyzed, and the derived formulas for determining the electrical impedance of piezoelectric elements were compared. Additionally, the dependence of the amplitude of electric current in the conductors of the circuit used for measuring the electrical impedance of a spherical piezoceramic element on frequency and the signs of the charge ratio on its polarized surfaces was studied.

The proposed problem of constructing a mathematical model of piezoelectric spherical shell actuators results in the determination of the dependence of electrical impedance on cyclic frequency and a set of parameters (geometric, physics and mechanical, electrical) for such actuators.

The obtained analytical dependencies, which can determine the electrical impedance and the amplitude values of current and electric charge on the electrode surface of the piezoelectric spherical shell actuator under the conditions of the reverse piezoelectric effect, allow for the complete solution of the problem of harmonic axisymmetric oscillations of a piezoelectric spherical shell of a specific diameter.

A frequency-dependent change in the quality factor and electrical impedance of the spherical shell model of the PZT-19 piezoceramic actuator was established, which oscillates. This revealed a higher quality factor (2-2.5 times higher compared to disk actuators and 1.4-1.8 times higher compared to cylindrical actuators) and a sharp decrease in such impedance (2.5-7 times) compared to cylindrical and disk actuators, occurring at electromechanical resonance frequencies around 2.4 kHz.

The evaluation of the efficiency of the spherical actuator compared to actuators of other shapes showed that these spherical actuators have 20-30 % better resistance to deformation compared to others, reducing the likelihood of unwanted oscillations and improving operational stability. Additionally, spherical actuators exhibit a 15-25 % lower tendency towards local resonances compared to disk and cylindrical actuators, which positively impacts the quality and accuracy of vibration generation.

The work presents data obtained as a result of the experimental scientific and technical development "Development of an automated ultrasonic system for extracting plant raw materials in the production of multi-nutrient functional beverages for the rehabilitation and prevention of post-traumatic stress disorders" (state registration number: 0124U000713, 2024-2025), conducted at the Cherkasy State Technological University.

Acknowledgements

The authors have not disclosed any funding.

Data availability

The datasets generated during and/or analyzed during the current study are available from the corresponding author on reasonable request.

Conflict of interest

The authors declare that they have no conflict of interest.

References

- **[1]** W. Wang, Y. Jiang, and P. J. Thomas, "Structural design and physical mechanism of axial and radial sandwich resonators with piezoelectric ceramics: a review," *Sensors*, Vol. 21, No. 4, p. 1112, Feb. 2021, https://doi.org/10.3390/s21041112
- **[2]** B. Wen, X. Huang, Y. Li, and Y. Zhang, *Vibration Utilization Engineering*. Singapore: Springer Nature Singapore, 2022, https://doi.org/10.1007/978-981-19-0672-5
- **[3]** K. Wang, Y. Yu, and A. Preumont, "Wavefront control strategies for large active thin shell primaries with unimorph actuators," *Actuators*, Vol. 12, No. 3, p. 100, Feb. 2023, https://doi.org/10.3390/act12030100
- **[4]** M. K. Kwak, *Dynamic Modeling and Active Vibration Control of Structures*. Dordrecht: Springer Netherlands, 2022, https://doi.org/10.1007/978-94-024-2120-0
- **[5]** S. M. Hosseini, H. Kalhori, and A. Al-Jumaily, "Active vibration control in human forearm model using paired piezoelectric sensor and actuator," *Journal of Vibration and Control*, Vol. 27, No. 19-20, pp. 2231–2242, Sep. 2020, https://doi.org/10.1177/1077546320957533
- **[6]** C. Bazilo, "Modelling of bimorph piezoelectric elements for biomedical devices," *Advances in Intelligent Systems and Computing*, Vol. 1126, pp. 151–160, Jan. 2020, https://doi.org/10.1007/978-3- 030-39162-1_14
- **[7]** R. Baraniuk and W.-G. Drossel, "Simplification of the model of piezoelectric actuator control based on preliminary measurements," *Actuators*, Vol. 9, No. 3, p. 90, Sep. 2020, https://doi.org/10.3390/act9030090
- **[8]** M. Kanchan, M. Santhya, R. Bhat, and N. Naik, "Application of modeling and control approaches of piezoelectric actuators: A review," *Technologies*, Vol. 11, No. 6, p. 155, Nov. 2023, https://doi.org/10.3390/technologies11060155
- **[9]** H. Hu, Z. Li, and X. Wang, "Modelling and analysis of piezoelectric actuators with partially debonded adhesive layers," *Mathematics and Mechanics of Solids*, Vol. 26, No. 5, pp. 722–737, Oct. 2020, https://doi.org/10.1177/1081286520966032
- **[10]** B. S. Aronov, *Piezoelectric Electromechanical Transducers for Underwater Sound. Part I. Introduction to Energy Method of Treating the Transducers*. Boston, USA: Springer, 2022.
- **[11]** V. V. Medianyk, Y. Y. Bondarenko, C. V. Bazilo, and M. O. Bondarenko, "Research of currentconducting electrodes of elements from piezoelectric ceramics modified by the low-energy ribbonshaped electron stream," *Journal of Nano and Electronic Physics*, Vol. 10, No. 6, pp. 6001–6012, 2018.
- **[12]** V. S. Antonyuk, M. O. Bondarenko, and Y. Y. Bondarenko, "Studies of thin wear-resistant carbon coatings and structures formed by thermal evaporation in a vacuum on piezoceramic materials," *Journal of Superhard Materials*, Vol. 34, No. 4, pp. 248–255, Aug. 2012, https://doi.org/10.3103/s1063457612040065
- **[13]** O. N. Petrishchev and C. V. Bazilo, "Methodology of determination of physical and mechanical parame-ters of piezoelectric ceramics," *Journal of Nano and Electronic Physics*, Vol. 9, No. 3, 2017.
- **[14]** C. Bazilo, I. Bondarenko, and L. Usyk, "Mathematical modeling of spherical piezoelectric elements of ultrasonic sensors for diagnostics of urban infrastructure components," in *Lecture Notes in Networks and Systems*, Vol. 808, Cham: Springer Nature Switzerland, 2023, pp. 283–293, https://doi.org/10.1007/978-3-031-46877-3_25
- **[15]** C. Bazilo, A. Zagorskis, O. Petrishchev, Y. Bondarenko, V. Zaika, and Y. Petrushko, "Modelling of piezoelectric transducers for environmental monitoring," in *10th International Conference "Environmental Engineering"*, 2017.
- **[16]** A. Buchacz, M. Placzek, and A. Wrobel, "Modelling of passive vibration damping using piezoelectric transducers – the mathematical model," *Maintenance and Reliability*, Vol. 16, No. 2, pp. 301–306, 2014.