Numerical simulation of the supercritical carbon dioxide flow in converted marine pipelines

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Abstract. The article discusses the features of modeling the flow of supercritical carbon dioxide in offshore pipelines of complex geometry. The concept is based on the replacement of the initial oil and gas in field pipelines for the re-injection of carbon dioxide into the reservoir with the purpose of further disposal. Various variants of the component composition are considered depending on the origin of the fluid. Needed initial operating parameters for conversion are defined.

Keywords: carbon capture storage and utilization, marine pipelines, converted pipelines, numerical modelling.

1. Introduction

Reducing the level of carbon dioxide emissions into the atmosphere is one of the priorities of the energy sector of developed countries. One of the ways to solve this issue is the direct capture of harmful substances from factories and industrial facilities. There are three types of capture, depending on the stage at which the collection takes place: pre-combustion, post-combustion and oxy-fuel. The degree of purity of the captured carbon dioxide and the component composition of the resulting gas depend on the chosen method [1].

The most common method of utilization of captured carbon dioxide is its injection into depleted deposits and salt caverns [2, 3]. The choice of the most suitable phase state for injection is determined by the conditions of a particular production facility and the existing infrastructure. The preferred conditions are pumping in the supercritical phase with pre-drying in order to minimize the risks of corrosion on the metal of the pipe. In this case, an important task is to determine the critical parameters of the fluid, which strongly depend on the components included in the composition [4]. An example of a phase envelope for post-combustion capture is shown in Fig. 1. Vapour-Liquid Equilibrium and Water lines are depicted as wells as Critical Point (C) and Dew Point (D).

To ensure pumping in a supercritical state, the fluid pressure and temperature must always be in the zone to the right of point C. Within the framework of this work, pumping through the existing offshore pipeline connecting the offshore platform and the coastal complex is considered. Thus, the conversion of the original pipeline is ensured. The geometry of the pipeline is presented at Fig. 2. The pipeline has internal diameter of 327 mm and the wall thickness of 14,5 mm and two extra layers: 6 mm of bitumen enamel to isolate the steel and 71 mm of concrete to fix the position. It is laid at the bottom of the trench with the average 1 m of burial. The 1:0 slope and cofferdam wall construction method are used. The total length is 17,5 km. The flow rate in the pipe is determined by the level of carbon dioxide captured from hydrogen production plants located near the platform.

Fig. 2. The studied marine pipeline geometry

2. Calculation model

The question of the correctness of the calculation of pressure losses in multiphase and supercritical pipelines laid over rough terrain was considered by many authors [5, 6]. Summing up the information, it can be concluded that the pressure drop is calculated by the following formula:

$$
\Delta P = \Delta P_{fr} + \Delta P_{ac} + \Delta P_g,\tag{1}
$$

where ΔP_{fr} – friction pressure drop, ΔP_{ac} – acceleration pressure drop and ΔP_g – pressure drop due to the gravity.

The named components are calculated by the following formulas:

$$
\Delta P_{fr} = \frac{G^2}{2\rho} \cdot \frac{L}{D} \cdot f,\tag{2}
$$

$$
\Delta P_{ac} = \frac{G^2}{2\rho} \cdot \frac{L}{D} \cdot f_{ac},\tag{3}
$$

$$
\Delta P_g = \pm g \left(\frac{\rho_{out} + \rho_{in}}{2} \right) \cdot L \cdot \sin \theta, \tag{4}
$$

where G – mass flux, kg/(m²·s); ρ – fluid density, kg/m³; L – tube length, m; D – inner diameter, m; f – friction factor; f_{ac} – acceleration factor; θ – pipe inclination angle.

The sign "+" is for the upward flow, and the sign "-" is for the downward flow.

The supercritical friction factor f is not to be confused with the friction coefficient, sometimes called the Fanning friction factor:

$$
f = f_{iso,b} \left(\frac{\mu_w}{\mu_b}\right)^{0.49 \left(\frac{\rho_f}{\rho_{pc}}\right)^{1.31}},\tag{5}
$$

where $f_{iso,b}$ – isothermal single-phase friction factor at fluid bulk temperature; μ_w , μ_b – dynamic viscosity at inner wall temperature and fluid bulk temperature respectively, Pa⋅s; ρ_f , ρ_{pc} – fluid density at film temperature and pseudo-critical temperature respectively, $kg/m³$.

For cooling processes, f_{ac} can be approximated with the one-dimensional approximation mode:

$$
f_{ac} \approx 2 \cdot D \cdot \rho_b \frac{d}{dL} \left(\frac{1}{\rho_b}\right) = \frac{-8q}{G} \left(\frac{\beta}{C_p}\right)_b,
$$
\n(6)

where ρ_b – fluid density at fluid bulk temperature, kg/m³; q – heat flux from fluid to wall, W/m²; β – thermal expansion coefficient at fluid bulk temperature, $1/K$; C_p – specific heat at constant pressure, J/(kg∙K).

The calculated dependences and equations created for modeling the fluid flow process in the pipeline can be divided depending on the method of their development. Empirical and mechanistic models are distinguished. According to the general principles of modeling, the first models contain many dimensionless complexes, but this imposes certain restrictions on their applicability to real problems (Beggs&Brill, Orkiszewski). On the other hand, mechanistic ones are related to the direct solution of the equation of fluid motion (Tulsa Unified Model, OLGAS, HTFS). Despite their high accuracy, the simulation results may still be unstable when simulating conditions beyond the initial working area. In this study, the mechanistic approach is applied.

Fig. 3. Pressure distribution along the pipeline at $\tau = 0$ (model 1)

Fig. 4. Pressure distribution along the pipeline at $\tau = 0$ (model 2)

Steady state calculation of pressure and temperature distribution was made using mechanistic approach. The case was to find the proper pressure and temperature at inlet if the same parameters at the outlet were known and provide SCF state. The pressure at the outlet was set at 7,64 MPa to cover the maximum critical value, the temperature was set at 32 °C (Fig. 3-6). The outlet temperature for pre-combustion stream was set at 33 °C due to convergence limitations. Two different correlations were applied to compare the results.

Fig. 5. Temperature distribution along the pipeline at $\tau = 0$ (model 1)

Fig. 6. Temperature distribution along the pipeline at $\tau = 0$ (model 2)

The resulting graphs at initial moment of time $(\tau = 0)$ clearly show a significant difference in the behavior of streams of different origin. Pressure and temperature levels for the pre-combustion flow are much higher than the ones for oxy-fuel and post-combustion due to the relatively high degree of contamination (Table 1). Despite the named aspect, all the cases can be implemented due to the fact that the maximum operating pressure (MOP) and maximum allowable operating pressure (MAOP) levels were not exceeded. Reference values are 9,1/9,8 MPa and 10 MPa respectively. However, the obtained values of temperature for pre-combustion stream are much higher than mean operating temperature of oil or gas. Reference values are 12,5 °C and 5 °C respectively. The results of modelling are presented at Table 1.

Analyzing the results, it can be established that the second calculation model shows larger values. However, the inlet temperature for pre-combustion is slightly lower for this correlation.

The resulting SCF flows are not stable if the change of parameters in time is considered. To track changes in the state of the system over time, the second model was applied due to its high efficiency for dynamic tasks. The problem was solved for a period of 72 hours. Results of modelling at certain points of time are presented at Figs. 7-12.

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Fig. 7. Pressure distribution for the pre-combustion flow

10000 Pipeline length, m **Fig. 9.** Pressure distribution for the oxy-fuel flow

12000

14000

16000

8000

7,53

200

4000

6000

According to the received schedules of changes, the following conclusions can be drawn. The oxy-fuel flow is subject to the greatest fluctuations: this is especially noticeable at 72 hours after the start of the simulation, when the pressure in the pipeline drops significantly along its entire length, and the temperature curve acquires a wave character at the point of local elevation of the route. In the latter case, there is a drop in the operating temperature below the critical value, which leads to the appearance of a liquid phase. As for the post-combustion flow, fluctuations in pressure also occurred, especially in the shallowest part of the pipeline route. The pre-combustion one turned out to be the most stable flow due to the comparatively high temperature values. One of the possible ways to stabilize the flow is to increase the pumping speed to the recommended value of 2 m/s [7]. This can be achieved by increasing the mass flow of air in the case of $CO₂$ capture from nearby industrial facilities.

Fig. 11. Temperature distribution for the post-combustion flow

Fig. 12. Temperature distribution for the post-combustion flow

Taking into account all the data obtained, the following points should be noted. Among three considered streams of different origin, the most optimal for use is post-combustion. Pre-combustion stream requires extremely high temperatures, and the existing pipelines are initially not designed for such thermal loads. Oxy-fuel option is the least stable one, which can lead to a change in the flow pattern in the pipeline with the slightest change in operating parameters.

3. Conclusions

The calculation of the necessary parameters for pumping supercritical carbon dioxide remains a difficult task due to the lack of a significant experimental base with which the obtained simulation results should be correlated. At the moment, many studies are being conducted, the purpose of which is to determine the most accurate numerical correlation for the flow of a

supercritical fluid for any geometry feature. Taking into account the fact that more and more natural reservoirs will become depleted in the next 20-30 years, the issue of reconfiguring the existing infrastructure for back injection into the reservoir is extremely relevant. The issue of a comprehensive study of pipelines remains open, thereof attention should be paid to the influence of converted fluid on residual deposits, pumping equipment and other aspects of operation. If a universal conversion standard is developed, effective disposal will be possible yielding in achieved zero emission goals.

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Data availability

The datasets generated during and/or analyzed during the current study are available from the corresponding author on reasonable request.

Conflict of interest

The authors declare that they have no conflict of interest.

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